NEEDLE ROLLER CAGES FOR ENGINE CONNECTING RODS

•Needle Roller Cages for Big End

Needle Roller Cages for Small End



Structure and Features

IKO Needle Roller Cages for Engine Connecting Rods are bearings for use in engine connecting rods. These bearings have superior performance proven in high performance engines of racing motor cycles, and are widely used in small motor vehicles, motor cycles, outboard marines, snow mobiles, high-speed compressors, etc. and also in general-purpose engines. Bearings for engine connecting rods are used under extremely severe and complex operating conditions such as heavy shock loads, high speeds, high temperatures and stringent lubrication.

Needle Roller Cages for Engine Connecting Rods are lightweight, and have high load ratings and high rigidity as well as superior wear resistance to withstand these severe conditions.

📕 Types

In Needle Roller Cages for Engine Connecting Rods, the types shown in Table 1 are available.

Table 1 Types

Туре	For big end	For small end		
Model code	KT…EG	KTV…EG		



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KT…EG

KTV…EG

Needle Roller Cages for Big End KT···EG

These roller cages are subjected to acceleration and deceleration during their rotating and epicyclic motion due to crank shaft rotation. To withstand such conditions, they are made of a special alloy and are lightweight with high rigidity.

They are guided on their outer periphery surface with superior lubricating properties.

For the purpose of using them under severe conditions such as high rotational speed and stringent lubrication, bearings plated with non-ferrous metals are also available on request.

High-load capacity and high-rigidity cages to be used for racing motor cycles (See the photo bellow.), split needle cages for solid (one-piece) type crank-shafts and other special specification cages of various types are also available. Please consult IKO when reauired.



High-load capacity and high-rigidity cage KTZ ··· EG

Needle Roller Cages for Small End KTV···EG

These roller cages oscillates at high speeds within a limited loading zone under heavy shock loads. Thus, these cages are designed to be lightweight and have high rigidity with a well-balanced structure. In these cages, a number of needle rollers having a small diameter are incorporated to reduce the rolling contact stress in the loading zone.

Needle Roller Cages for Small End are classified into two types, the outer surface guide type and the inner surface guide type. This classification is shown in the table of dimensions.

In the outer surface guide type, the cage is guided by the sliding contact between the inner surface of the connecting rod and the outer surface of the cage. In the inner surface guide type, the cage is guided by the sliding contact between the outer surface of the pin and the inner surface of the cage.

Identification Number

The identification number of Needle Roller Cages for Engine Connecting Rods consists of a model code. dimensions and any supplemental codes as shown below.

f identification	number	
e Dimensions		Supplemental code
22 28 16	EG	B2
]	\mathcal{I}	
diameter 1)		
de diameter m)		
]		
nean value of e Table 2.)		
	Dimensions 22 28 16 diameter n) de diameter m) nean value of	Dimensions 22 28 16 EG diameter n) de diameter m) nean value of

Accuracy

The diameter tolerances of needle rollers of Needle Roller Cages for Engine Connecting Rods are classified as shown in Table 2. When the classification symbol is not indicated in the identification number, the classification symbol "B2" is applied.

The tolerance of the cage width B_c is $-0.2 \sim -0.4$ mm. But cages with marks in the B_c column in the dimension tables are manufactured with the following width tolerances.

●:0~-0.2 mm ■:·	$-0.1 \sim -0.3$ mm
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Table 2 Tolerances of needle roller diameter unit[.] // m

Class	Classification symbol(¹)	Tolerance of mean value of roller dia. (²)
Standard	B 2 B 4	$0 \sim -2 - 2 - 2 \sim -4$
Semi-standard	B 6 B 8 B10	$ \begin{array}{r} -4 & \sim -6 \\ -6 & \sim -8 \\ -8 & \sim -10 \end{array} $

Notes(1) The classification symbol is indicated at the end of the identification number

(2) Tolerances for circularity are based on JIS B 1506 (Rolling bearings - Rollers).

Clearance

Radial internal clearances are selected according to the type of engine and the operating conditions (rotational speed, load, lubricating conditions, etc.). If a bearing is used with an inadequate clearance, bearing troubles such as seizure, early flaking and noise increase may occur, leading to an engine failure. Therefore, it is necessary to select the clearance carefully according to test results and experience.

Recommended radial internal clearances are shown in Table 3. When operating at high speeds, it is recommended to select the upper limit of the clearance.

Fit

To obtain the recommended clearance shown in Table 3, it is general practice to match a connecting rod, crank pin or piston pin and needle roller cage of suitable tolerances for assembly.

Precautions for Use

When designing a connecting rod, crank pin and piston pin, the following precautions should be taken, because the raceways are subjected to loads under extremely severe conditions.

Material

It is recommended to use carburizing steel because the raceways are subjected to fluctuating loads with frequent and heavy shock loads. Generally, chromium molybdenum steel is used. Nickel chromium molybdenum steel is also used.

2 Hardness

The recommended surface hardness of the raceway is 697~800HV (60~64HRC). While the effective hardening depth differs depending on the applications, the general value is $0.6 \sim 1.2$ mm.

Surface roughness

To minimize initial wear and to extend life, it is recommended that the surface roughness of the crank pin and piston pin be $0.1 \mu m R_a$ or less, and the surface roughness of the connecting rod large end and small end bores be $0.2 \mu mR_{\odot}$ or less.

4 Accuracy

Circularity and cylindricity of connecting rod, piston pin and crank pin are as shown in Table 4.

6 Parallelism and torsional accuracy of connecting rod bores

 $L\pm0.02$ mm and $E\pm0.02$ mm shown in Fig. 1 indicate the parallelism and torsional accuracy between the big end and small end bores of the connecting rod, respectively. The tolerance range is 0.04 mm or less per 100 mm in case of a general-purpose engine and 0.02 mm or less for a high-speed engine such as a racing motorcycle engine. When these accuracy conditions are not satisfied, the axial forces on the needle roller cage and connecting rod will increase, directly leading to a failure such as seizure. Careful consideration is required.

Table 3 Recommended radial internal clearance

			unit. µ m
Shaf m		Big end	Small end
Over	Incl.		
_	18	$(d_{\rm p}-6)\sim d_{\rm p}$	
18	30	$(d_{\rm p}^{\rm P} - 8) \sim d_{\rm p}^{\rm P}$	$3{\sim}15$
30	40	$(d_{ m p}-$ 12) $\sim d_{ m p}$	

Remark $d_{\rm p}$ is obtained using the following equation for roller pitch circle diameter in millimeters, and changing the unit from millimeters to micrometers Roller pitch circle dia. = $F_{\rm w} + E_{\rm w}$ Example KT 222814 EG for big end

Recommended clearance is: $17 \sim 25 \ \mu$ m

Table 4 Accuracy of connecting rod, piston pin

and crank pin unit: µ										
Range m		Crank pin o Piston pin o	liameter d_1 diameter d_2	Big end bore $D_1 \\ {\rm Small} \ {\rm end} \ {\rm bore} \ D_2 \\$						
Over Incl.		Circularity MAX.	Cylindricity MAX.	Circularity MAX.	Cylindricity MAX.					
-	18	1	2	2	3					
18	30	2	3	3	4					
30	40	3	4	4	5					

Remark Refer to Fig.1 for the dimension symbols.



С

KT…EG

KTV…EG

Needle Roller Cages for Big End





KT ··· EG

Shaft dia. 8 – 32mm

Shaft		Mass (Ref.)	Bounda	ary dime mm	ensions	Basic dynamic load rating	Basic static load rating
dia.	Identification number					С	C_0
mm		g	$F_{\rm w}$	$E_{\rm w}$	B _c	N	N
8	KT 8128 EG	2.1	8	12	8	3 280	2 660
10	KT 101410 EG	3.2	10	14	10	4 900	4 680
12	KT 121610 EG KT 121710 EG	3.8 5.3	12 12	16 17	10 10	5 650 6 670	5 890 6 380
14	KT 14199.7 EG KT 141910 EG	5.7 5.7	14 14	19 19	9.7 10	6 120 6 640	5 880 6 530
15	KT 15199 EG KT 152010 EG	4.2 6.1	15 15	19 20	9 10	5 790 7 100	6 460 7 260
16	KT 162211.5 EG KT 162212 EG	9.5 9.7	16 16	22 22	■11.5 12	9 550 10 500	9 660 10 900
18	KT 182210 EG KT 182411.6 EG KT 182412 EG	5.7 11 11	18 18 18	22 24 24	10 ■11.6 12	7 500 10 600 11 800	9 560 11 500 13 100
20	KT 202612 EG KT 202614 EG KT 202814 EG	12 13.8 20	20 20 20	26 26 28	12 14 •14	12 400 13 000 15 700	14 300 15 200 16 100
22	KT 222814 EG KT 222816 EG KT 222912 EG KT 223215 EG	14.9 17.5 15.2 30	22 22 22 22 22	28 28 29 32	14 16 12 15	13 600 15 700 12 900 21 300	16 600 19 800 14 000 21 500
23	KT 232913 EG	14.9	23	29	13	12 800	15 600
24	KT 243015 EG KT 243016 EG	17.9 18.2	24 24 24	30 30	15 16	14 200 16 300	18 000 21 500 26 400
00	KT 243120 EG	28		31	20	20 800	26 400
30	KT 303818 EG	35.5	30	38	18	24 900	32 600
32	KT 324220 EG	54	32	42	20	31 900	39 400

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IKO

KTV ··· EG

(Shaft dia. 9 – 18mm)

Shaft dia.	Identification number	Mass (Ref.)			Basic dynamic load rating C	Basic static load rating C_0	Cage guide type	
mm		g	$F_{\rm w}$	$E_{\rm w}$	B _c	N	N	
9	KTV 91211.5 EG	2.8	9	12	•11.5	3 900	4 070	Outer surface guide
	KTV 91214 EG	3.5	9	12	14	4 440	4 810	Inner surface guide
10	KTV 101316 EG	4.5	10	13	16	4 400	4 880	Inner surface guide
	KTV 101410 EG	3.8	10	14	10	4 520	4 220	Inner surface guide
	KTV 101411 EG	4.1	10	14	11	5 060	4 880	Outer surface guide
	KTV 101412.5 EG	4.8	10	14	• 12.5	5 590	5 540	Inner surface guide
10.5	KTV 10.51415 EG	5.1	10.5	14	15	5 710	6 270	Outer surface guide
12	KTV 121514.3 EG	4.3	12	15	•14.3	5 840	7 390	Outer surface guide
	KTV 121613 EG	5.6	12	16	13	7 020	7 800	Outer surface guide
	KTV 121615.5 EG	6.8	12	16	•15.5	7 600	8 600	Outer surface guide
14	KTV 141812 EG	6	14	18	12	6 780	7 760	Inner surface guide
	KTV 141816.5 EG	8.2	14	18	16.5	9 180	11 500	Outer surface guide
	KTV 141822 EG	10.8	14	18	•22	9 950	12 600	Inner surface guide
16	KTV 162019 EG	10.6	16	20	19	10 800	14 600	Outer surface guide
	KTV 162022 EG	12.7	16	20	22	11 400	15 700	Inner surface guide
18	KTV 182223.5 EG	14.9	18	22	■23.5	13 000	19 300	Inner surface guide
	KTV 182321 EG	16.4	18	23	21	14 400	18 900	Inner surface guide